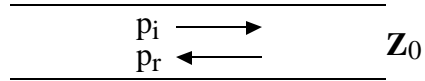


(10.10) Reflection and Transmission of Waves in a Pipe

General case: Acoustic impedance discontinuity at $x = 0$



For a pipe,

$$Z = \frac{\mathbf{r}_0 c}{S} \quad (\text{for a traveling wave})$$

and

$$p_i = A e^{j(\omega t - kx)}$$

$$p_r = B e^{j(\omega t + kx)}$$

for the standing wave case

$$\tilde{Z} = \frac{p_i + p_r}{U_i + U_r} = \frac{\mathbf{r}_0 c}{S} \frac{\tilde{A} e^{-jkx} + \tilde{B} e^{jkx}}{\tilde{A} e^{-jkx} - \tilde{B} e^{jkx}}$$

$$\tilde{Z} = Z_0 = \frac{\mathbf{r}_0 c}{S} \frac{\tilde{A} + \tilde{B}}{\tilde{A} - \tilde{B}} \quad @ \text{ interface } (x=0)$$

$$\frac{\tilde{B}}{\tilde{A}} = \frac{Z_0 - \frac{\mathbf{r}_0 c}{S}}{Z_0 + \frac{\mathbf{r}_0 c}{S}}$$

For $\tilde{Z}_0 = R_0 + jX_0$ the power reflection and transmission coefficients are

$$R_p = \left| \frac{B}{A} \right|^2 = \frac{\left(R_0 - \frac{\mathbf{r}_0 c}{S} \right)^2 + X_0^2}{\left(R_0 + \frac{\mathbf{r}_0 c}{S} \right)^2 + X_0^2}$$

$$T_p = \frac{4R_0 \frac{\mathbf{r}_0 c}{S}}{\left(R_0 + \frac{\mathbf{r}_0 c}{S} \right)^2 + X_0^2}$$

Change in acoustic impedance can be associated with a change in fluid or a change in cross sectional area of the pipe without a change of medium (this can also come from a branch or a side port in a pipe).

Case 1: Simple change in cross-sectional area

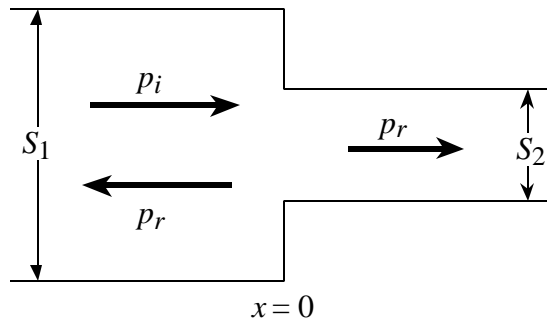


Figure 10.10.1 Transmission and reflection of a plane wave in the vicinity of a junction between two pipes where the cross-sectional area changes from S_1 to S_2 .

$$R_{\Pi} = \frac{(S_1 - S_2)^2}{(S_1 + S_2)^2}$$

$$T_{\Pi} = \frac{4S_1 S_2}{(S_1 + S_2)^2}$$

Case 2: Branch in pipe

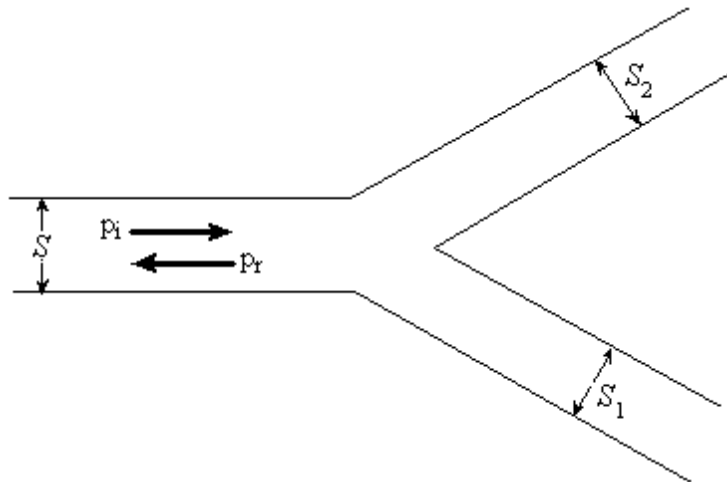


Figure 10.10.2 Conditions in the vicinity of a branch. The branches have cross-sectional areas S_1 and S_2 and input acoustic impedances Z_1 and Z_2 .

$$p_i = P_i e^{j\omega t}$$

$$p_r = P_r e^{j\omega t}$$

$$p_1 = Z_1 U_1 e^{j\omega t}$$

$$p_2 = Z_2 U_2 e^{j\omega t}$$

The boundary conditions at $x = 0$ (location of branch)

$$p_i + p_r = p_1 = p_2 \Leftrightarrow \text{cont. of pressure}$$

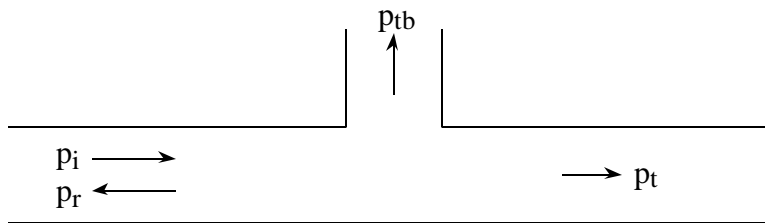
$$U_i + U_r = U_1 + U_2 \Leftrightarrow \text{cont. of volume velocity}$$

Application of these boundary conditions give

$$\frac{1}{Z_0} = \frac{1}{Z_1} + \frac{1}{Z_2} \quad \begin{array}{l} \text{admittances add} \\ \text{to give admittance} \\ \text{@ } x = 0 \end{array}$$

$$Z_0 = \frac{Z_1 Z_2}{Z_1 + Z_2} \quad \begin{array}{l} \text{If two impedances} \\ \text{in parallel} \end{array}$$

Case 3: Side branch



$$Z_b = R_b + j X_b$$

$$R_{\Pi} = \frac{\left(\frac{r_0 c}{2S}\right)^2}{\left(\frac{r_0 c}{2S} + R_b\right)^2 + X_b^2}$$

$$T_{\Pi} = \frac{R_b^2 + X_b^2}{\left(\frac{r_0 c}{2S} + R_b\right)^2 + X_b^2}$$

$$\begin{aligned} T_{\Pi b} &= 1 - R_{\Pi} - T_{\Pi} \\ &= \frac{\frac{r_0 c}{S} R_b}{\left(\frac{r_0 c}{2S} + R_b\right)^2 + X_b^2} \end{aligned}$$

Special Cases

$$R_b = X_b = 0$$

$$T_{\Pi} = 0$$

$$R_{\Pi} = 1$$

$$T_{\Pi b} = 0$$

R_b finite but not 0

Some power trans. into branch

$R_b = X_b = \infty$

$T_{\Pi} = 1$

As though branch is not there.

$R_{\Pi} = 0$

$T_{\Pi b} = 0$

$R_b = 0$ $X_b \neq 0$

No power into branch.

(Ex. branch is pipe with rigid termination)

X_b value does affect power trans. and reflected.

(10.11) Acoustic Filters

(a) Low-Pass Filters

Insert length L of pipe with different, S_1 , cross sectional area.

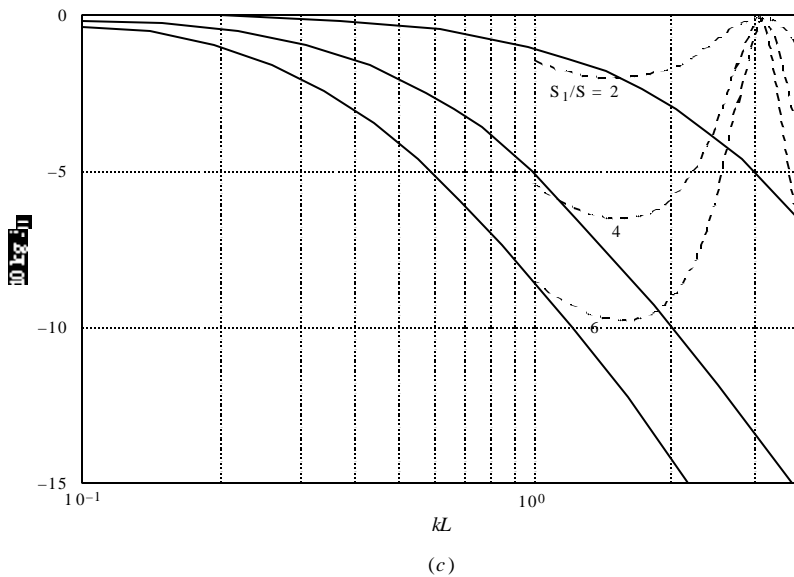
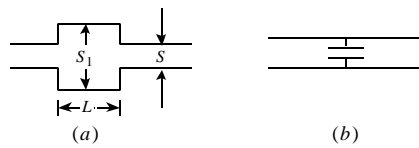


Figure 10.11.1 A simple low-pass acoustic filter consists of an enlarged section of cross-sectional area S_1 and length L in a pipe of cross-sectional area S .

(a) Schematic. (b) Analogous electric filter. (c) Attenuation for several values of S_1/S . Solid lines are from (10.11.2) for $kL \ll 1$. Dashed lines are from Problem 10.11.6 for $kL \gg 1$.

At low frequency $kL \ll 1$. The extra length of pipe acts like a side branch with

compliance $C = \frac{V}{r_0 c^2}$ and $V = (S_1 - S)L$ giving an acoustic impedance for the chamber of

$$Z_b \approx 0 - j \frac{r_0 c^2}{\omega (S_1 - S) L}$$

$$T_{\Pi} \approx \frac{1}{1 + \left(\frac{S_1 - S}{2S} kL \right)^2} \quad \begin{array}{l} \text{True ONLY} \\ \text{for } kL \ll 1 \end{array}$$

Actually, it acts similar to the transmission through a layer examined earlier in chapter 6

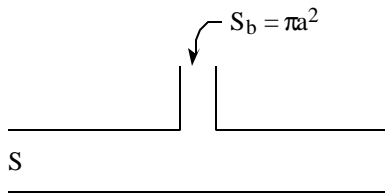
$$T_{\Pi} = \frac{4}{4 + \left(\frac{S_1}{S} - \frac{S}{S_1} \right)^2 \sin^2(kL)}$$

for small kL and for the middle layer being analogous to characteristic impedance of z_2 .

Like the layer transmission case, at higher frequency, the pipe will have frequencies for complete transmission $\left(L = m \frac{\lambda}{2} \right)$ and others where there is a minimum.

Used in mufflers, gun silencers, sound-absorbing plenum chambers in ventilation systems.

(b) High-Pass Filters



Short unflanged branch of radius a and length L . $L' = L + 1.4 a$

$$Z_b = \frac{r_0 c k^2}{4\rho} + j\omega \left(\frac{r_0 L'}{\rho a^2} \right) = R_b + jX_b$$

giving

$$T_{\Pi} = \frac{1}{1 + \left(\frac{\rho a^2}{2S L' k} \right)^2}$$

Most power is reflected at low frequencies. This implies that low frequencies are not transmitted down the pipe and radiated out (like a wind instrument).

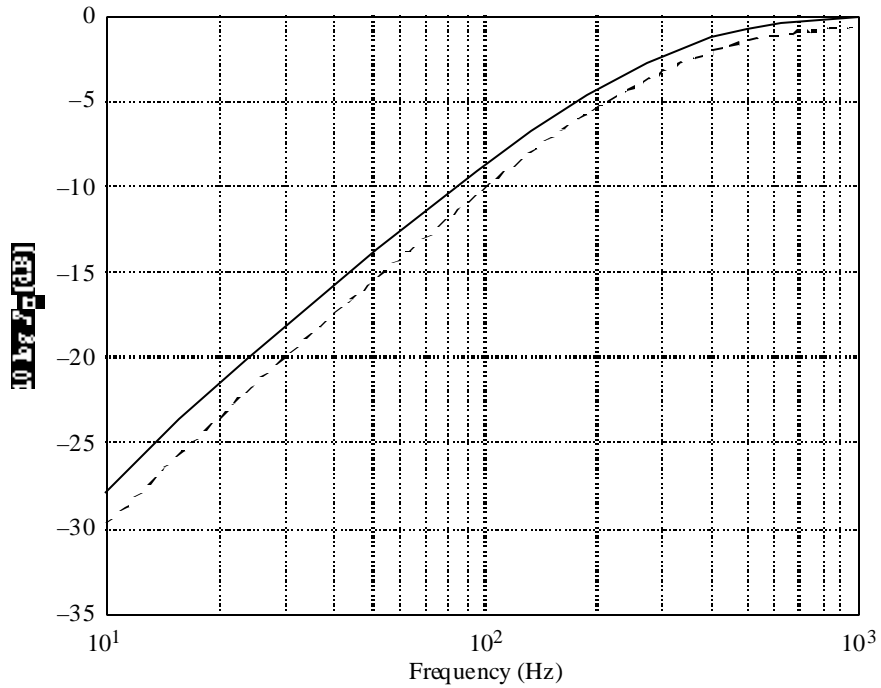
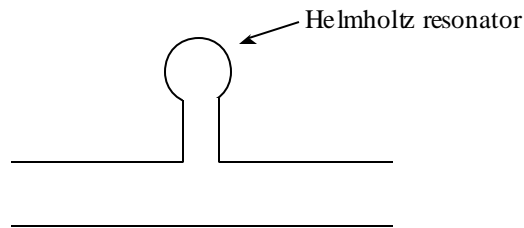


Figure 10.11.3 Attenuation for a high-pass filter in a pipe of cross-sectional area $S = 28 \text{ cm}^2$. The side branch has a 1.55 cm radius. The solid curve is for $L = 0.6 \text{ cm}$ and the dashed curve for $L = 0$.

(c) Band-Stop Filters



$$Z_b = 0 + jr_0 \left(\frac{\omega L'}{S_b} - \frac{c^2}{\omega V} \right)$$

$$T_{II} = \frac{1}{1 + \left(\frac{c/2S}{\frac{\omega L'}{S_b} - \frac{c^2}{\omega V}} \right)^2}$$

$$T_{II} = 0 \text{ for } \omega = c \sqrt{\frac{S_b}{L'V}} = \text{resonance freq. of Helmholtz resonator}$$

All energy is reflected \rightarrow none is lost in resonator.

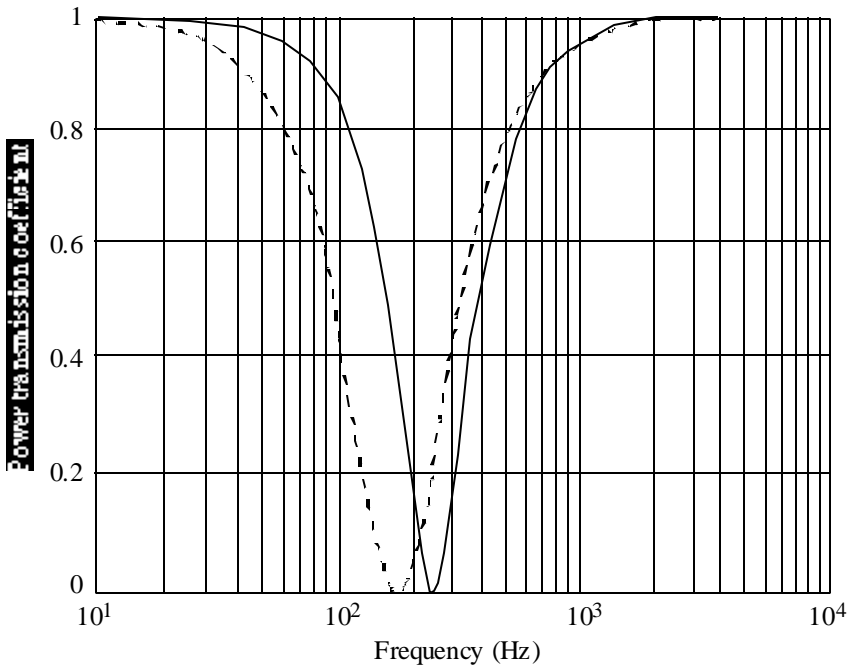


Figure 10.11.4 The power transmission coefficient for a band-stop filter consisting of a Helmholtz resonator. The resonator has a neck of length 0.6 cm and radius 1.55 cm. The pipe has a cross-sectional area 28 cm^2 . The solid line is for a resonator volume of 1120 cm^3 . The dashed line is for a resonator volume of 2240 cm^3 .